

## 1 Benchmark problem

The physical Model

$$\begin{cases} \partial_\tau(h) + \partial_\xi(\kappa h) & = -\alpha_\rho \mathcal{M} \\ \partial_\tau(hu) + \partial_\xi(hu \otimes \kappa + p) & = -\alpha_\rho u \mathcal{M} + h\mathcal{S}(u, \theta) \\ \partial_\tau x & = -(\theta - \Theta_n)\mathcal{E}(h, u, \theta)\boldsymbol{\eta}(\theta) \end{cases} \quad (1)$$

where  $h$  is the ,  $u$  is the fluid velocity,  $x(\xi, t)$  is the position interface position,  $\xi$  is the lagrangian coordinate,  $\Theta_n$  is the neutral angle,  $\theta = (\theta, \phi)$  is the local inclination angles,  $\boldsymbol{\eta}(\theta)$  is the local unit normal to the interface.  $\theta$  is the angle between the  $z$  axis and  $\boldsymbol{\eta}(\theta)$ ,  $\phi$  is the angle between the  $x$  axis and  $\boldsymbol{\eta}(\theta)$ .

$$\kappa(\theta) = \frac{u}{\cos \theta} \quad \text{and} \quad \boldsymbol{\eta}(\theta) = \begin{pmatrix} -\sin \theta \\ \cos \theta \end{pmatrix} \quad (2)$$

The pressure is defined according to the *Mohr – Coulomb* behavior

$$p(h, \theta, \kappa) = \frac{h^2 \beta(\kappa, \theta)}{2}, \quad \text{with} \quad \beta(\kappa, \theta) = (\cos^2 \theta) \mathcal{K}(\kappa)$$

where the factor  $\mathcal{K}(\kappa)$  is define, for  $\varepsilon = \frac{H_{ref}}{L_{ref}}$ , as

$$\mathcal{K}(\kappa) = \begin{cases} \mathcal{K}_-(\varepsilon, \theta_\phi, \theta_\delta) & \text{for } \partial_\xi \kappa \geq 0 \\ \mathcal{K}_+(\varepsilon, \theta_\phi, \theta_\delta) & \text{for } \partial_\xi \kappa < 0 \end{cases}$$

$$\mathcal{K}_\pm(\varepsilon, \theta_\phi, \theta_\delta) = \varepsilon \left( \frac{2 \left( 1 \pm \sqrt{1 - \frac{\cos^2 \theta_\phi}{\cos^2 \theta_\delta}} \right)}{\cos^2 \theta_\phi} - 1 \right)$$

where  $\theta_\phi$  and  $\theta_\delta$  are respectively the internal and the basal friction angles.

### 1.1 Model for the net driving acceleration

$$\mathcal{S}(u, \theta) = \cos \theta \left( \sin \theta - \varepsilon^\beta \text{sgn}(u) \mu \cos \theta \right)$$

where  $\mu = 0.9 * \tan(\theta_f)$  with  $\theta_f$  the friction angle.

## 1.2 Erosion/deposition

$$\mathcal{E}(h, u, \theta) = \mathcal{F}_e(h, u, \theta) \mathcal{F}_h(h)$$

$$\mathcal{F}_e(h, u, \theta) = \alpha_e \left( \frac{(1 - \tanh(e_\alpha(|\kappa| - \kappa_{th})))}{2} \mathcal{H}(\theta - \Theta_n) + \mathcal{H}(\Theta_n - \theta) \right)$$

Acordind to the relation  $\mathcal{H}(\Theta_n - \theta) = 1 - \mathcal{H}(\theta - \Theta_n)$ , we can also use the simplified form

$$\mathcal{F}_e(h, u, \theta) = \alpha_e \left( 1 - \frac{(1 + \tanh(e_\alpha(|\kappa| - \kappa_{th})))}{2} \mathcal{H}(\theta - \Theta_n) \right)$$

$$\mathcal{F}_h(h) = h + \alpha_h \sqrt{h}, \quad \kappa_{th} = \alpha_v (\theta - \Theta)^2$$

## 1.3 Benchmark Data

$$\theta_\phi = 34^\circ, \quad \Theta_n = 34^\circ, \quad \theta_\delta = 23^\circ, \quad \theta_f = 33^\circ$$

$$\alpha_h = 0.05, \quad \alpha_e = 2.0, \quad \alpha_v = 1.0, \quad \alpha_\rho = 0.9, \quad e_\alpha = 20, \quad \varepsilon = 1.0.$$

## 1.4 Initial condition

Computational domain  $[-3, 3]$ . Initial uniform mesh according to  $x$  and mesh sizes of : 100, 200, 400.

The initial basal surface

$$\mathcal{B}(\tau = 0, x) = \mathcal{B}_0 \exp\left(-\frac{32x^2}{9}\right)$$

The initial layer profile

$$h(\tau = 0, x) \cos \theta_0(x) = L_0 \exp\left(-\frac{32x^2}{9}\right) - \mathcal{B}_0 \exp\left(-\frac{32x^2}{9}\right)$$

$$L_0 = 2,$$

	<i>Test1</i>	<i>Test2</i>	<i>Test3</i>
$\mathcal{B}_0 =$	0	0.2	0.5